Section 5.3

Two dimensional oscillators Section 5.4 *Damped* oscillations

Read Sections 5.3 and 5.4.

Figure 5.7 (a) A restoring force that is proportional to r defines the isotropic harmonic oscillator. (b) The mass at the center of this arrangement of springs would experience a net force of the form $\mathbf{F} = -k\mathbf{r}$ as it moves in the plane of the four springs.

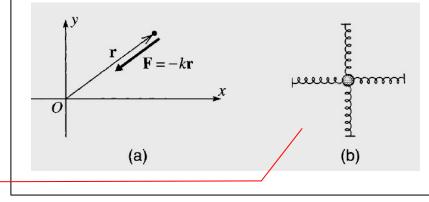
for small oscillations

5.3. Two dimensional oscillators The definition of an "isotropic" oscillator in 2 or 3 dimensions is F = -kr

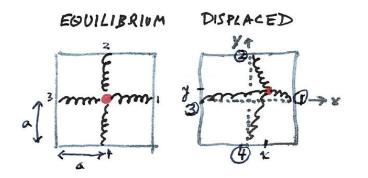
$$J = \frac{1}{2} k r^{2} = \frac{1}{2} k (x^{2} + y^{2} + z^{2})$$

in 3 dimensions

Figure 5.7 shows a 2d example; the particle (mass m) attached to the 4 springs moves in the xy plane.



<u>Comments about Figure 5.7.</u> The particle (mass = m) attached to the springs moves in the xy plane.



What is the potential energy when the particle is displaced to { x , y }?

Assume that the equilibrium length of each spring is *a*, and the spring constant is k/2. Also, the size of the square is

 $2a \times 2a$.

-0.5 10 0.0 0.5 ~ a-y+ 22 $U \approx \frac{1}{2} k (x^2 + y^2) \quad i = k^2 + y^2 \ll a^2$ = = + kr2 ; := = - 4F

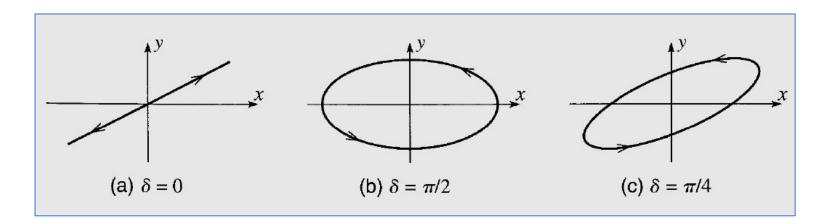
⁵ It is perhaps worth pointing out that one does *not* get a force of the form (5.17) by simply attaching a mass to a spring whose other end is anchored to the origin. 2

Figure 5.8. Three examples of *isotropic* oscillations in 2d:

i.e., $k_x = k_y$

$$U = \frac{1}{2} k x^{2} + \frac{1}{2} k y^{2}$$

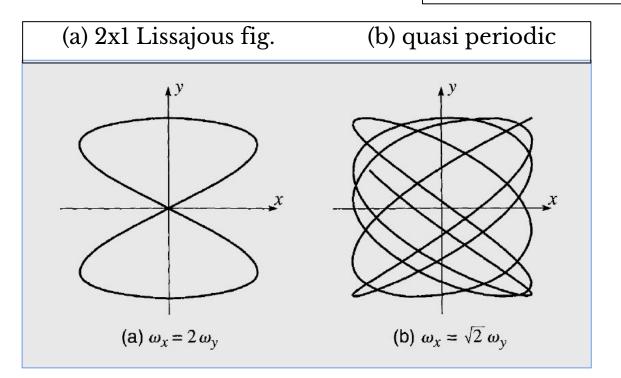
x(t) = A cos(wt)
y(t) = B cos(wt - δ)



<u>Figure 5.9.</u> Two examples of *anisotropic* oscillations *i.e.*, $k_x \neq k_y$

$$U = \frac{1}{2} k_x x^2 + \frac{1}{2} k_y y^2$$

x(t) = A cos(w_xt)
y(t) = B cos(w_yt - \delta)



4

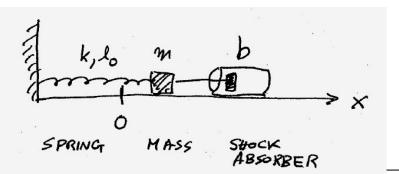
5.4. Damped oscillations

Sometimes in everyday life, oscillations may create problems.

For example, that's why a car has shock absorbers — to damp out the oscillations when the wheels hit a bump in the road, or a pothole.

Go back to 1-dimensional oscillations, but now add damping.

Generic picture



The equation of motion is

$$ma = -bv - kx$$

Note the assumption of "linear damping"; i.e., $F_{damping} = -b v$; or, we can write it this way, $m \ddot{x} + b \dot{x} + k x = 0$.

It is useful to *"rescale the parameters"* to write the equation in a standard form ;

$$x + 2\beta x + \omega_0^2 x = 0$$

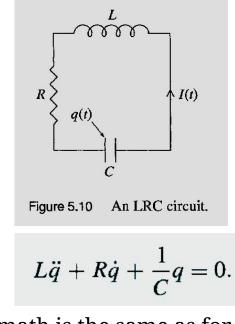
where

$$2\beta = b/m$$
 and $\omega_0^2 = k/m$.

<u>Figure 5.10</u>

THE EQUIVALENT LRC CIRCUIT

Recall from circuit theory



so the math is the same as for the mechanical system.

$$x + 2 \beta x + \omega_0^2 x = 0$$

Solution. This is an example of a *"homogeneous linear differential equation with constant coefficients"*. There is a standard method to solve this kind of diff. eq. (MTH 234)

First, try
$$\chi(t) = e^{pt}$$
.
 $\dot{\chi} = p e^{pt}$ and $\ddot{\chi} = p^2 e^{pt}$, so
 $p^2 + 2\beta p + \omega_o^2 = 0$
 $p_{\pm}^2 = -\beta \pm \sqrt{\beta^2 - \omega_o^2}$

6

$$x'' + 2 \beta x' + \omega_0^2 x = 0$$

• We have two solutions, $exp(p_t)$ and $exp(p_t)$ where

 $p_{\pm} = -\beta \pm \sqrt{\beta^2 - \omega_0^2}$

■ The equation is second order, so the *general solution* depends on two constants. The equation is linear so we can write the general solution as

$$\chi(t) = c_{+}e^{\beta_{+}t} + c_{-}e^{\beta_{-}t}$$
$$= e^{-\beta_{+}t} \left\{ c_{+}e^{\sqrt{\beta_{-}\omega_{0}^{2}}t} + c_{-}e^{-\sqrt{\beta_{-}^{2}\omega_{0}^{2}}t} \right\}$$

The 2 constants, c_{+} and c_{-} , must be determined from the initial conditions or some other information.

<u>Overdamped oscillator</u>; $\beta > \omega_0$ This is the case of *strong* damping. In this case p₁ and p₂ are *real*. $\chi(o) = C_{+} + C_{-}$ and $U(o) = P_{+}C_{+} + P_{-}C_{-}$ $C_{\pm} = \left[p_{\pm} \chi_{(0)} - \sigma_{(0)} \right] / \left(p_{\pm} - p_{\pm} \right)$ <u>Underdamped oscillator</u>; $\beta < \omega_0$ This is the case of *weak* damping. In this case p_1 and p_2 are complex numbers. Recall $e^{\pm i\theta} = \cos\theta \pm i\sin\theta$ (Euler) $\chi(t) = e^{-\beta t} \left[A \cos \omega t + B \sin \omega_t t \right]$ where $\omega_1 = \sqrt{\omega_0^2 - \beta^2}$. $\chi(o) = A$ and $\dot{\chi}(o) = -BA + co, B$

7

The critically damped oscillator

<mark>β = ω_ο</mark>

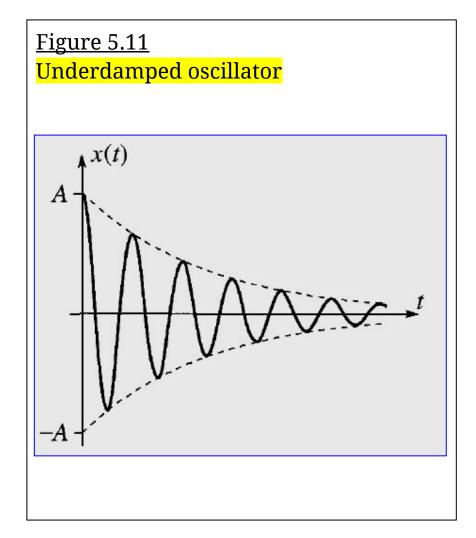
In this case p_+ and p_- are *equal*, $p_+ = p_- = \omega_0$; so exp(pt) is only one solution. To get the general solution we need another solution.

<u>Exercise</u>: Show that x(t) = t exp(pt) is also a solution for the critically damped oscillator $(\beta = \omega_0)$.

$$\chi(t) = e^{-\beta t} [A + Bt]$$

$$\chi(t) = A \text{ and } \dot{\chi}(t) = -\beta A + B$$

Example. Consider these initial conditions: x(0) = 1 and v(0) = 0. X .0 0.8 0.6 $\beta = 2 \omega_0$ $\beta = \omega_0$ 0.4 $s = 0.5 \omega_0$ 0.2 0.2 04 0.6 0.8 1.0 1.2 1.4



<u>Figure 5.12</u>

Overdamped oscillator

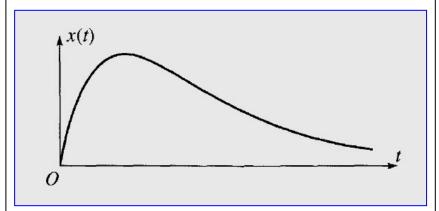
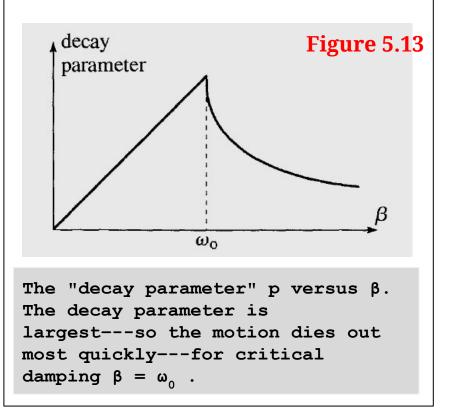


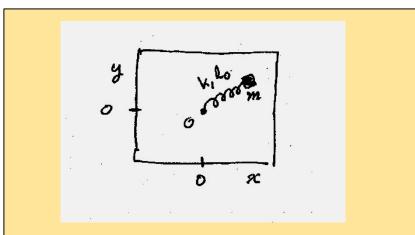
Fig. 5.12 corresponds to these initial conditions: x(0) = 0 and v(0) > 0; i.e., 1- the mass is kicked in the +x direction, 2- it reaches a maximum displacement, and 3 - it returns to equilibrium monotonically.

<u>Critical damping</u> $(\beta = \omega_0)$ This special case has the most rapid

return to equilibrium ...



none under	$\beta = 0$ $\beta < \omega_{\rm o}$	decay parameter 0 β	
critical over	$\beta = \omega_{\rm o}$ $\beta > \omega_{\rm o}$	$\beta - \sqrt{\beta^2 - \omega_o^2}$	



A mass m moves in the xy-plane, attached to a spring as shown. According to a footnote in Taylor, the force on m is not -kr.

OK, then, what *is* the force?

Homework Assignment #9 due in class Wednesday November 1 [41] Problem 4.41 and Problem 4.43 [42] SEE THE COVER SHEET [43] Problem 5.3 * [44] Problem 5.5 * [45] Problem 5.9 * [46] Problem 5.12 ** [47] Problem 5.18 ***

Use the cover sheet.